

IN THE SPECIFICATION

Page 7, line 8 through page 8, line 2 have been amended as follows:

A switch member 40 is provided for controlling position of the pawl 30 in the compartment 14. The switch member 40 includes a substantially cylindrical body 47 that is rotatably received in the compartment 14 and a turn piece 41 that extends outward from an end of the cylindrical body 47 to a position beyond the handle 12 via the opening 15 of the handle 12 for manual operation. The cylindrical body 47 includes a receptacle 42 for receiving an elastic element 46 and a pressing member 45 having a receptacle 451 defined therein. As illustrated in Fig. 3, the pressing member 45 is partially received in the receptacle 42 of the cylindrical body 47, with an end of the elastic element 46 attached to an end wall delimiting the receptacle 42 of the cylindrical body 47 and with the other end of the elastic element 46 attached to an end wall delimiting the receptacle 451 of the pressing member 45. The pressing member 45 is normally biased by the elastic element 46 to press against one of the inclined faces 341 and 342 of the pawl ~~[[34]]~~ **30** (e.g., the inclined face 342, see Fig. 3A), thereby urging a portion of the teeth 31 of the pawl 30 to engage with the teeth 21 of the drive member 20. In this case, as shown in Fig. 3A, the wrench allows ratcheting operation (i.e., tightening or loosening of a fastener) in the counterclockwise direction and allows free rotation in the clockwise direction (i.e., the fastener is not turned when the handle 12 is turned clockwise). It is noted that the abutting face 33 of the pawl 30 abuts a wall delimiting the compartment 14 of the handle 12 when the drive member 20 is turned in the ratcheting direction.

Page 8, lines 3-14 have been amended as follows:

When the turn piece 41 of the switch member 40 is turned, the pressing member 45 is moved from the inclined face 342 to the other inclined face 341. The other portion of the teeth 31 of the pawl 30 engages with the teeth 21 of the drive member 20. In this case, the wrench allows ratcheting operation in the clockwise direction and allows free rotation in the counterclockwise direction (i.e., the fastener is not turned when the handle 12 is turned counterclockwise). It is noted that the abutting face 32 of the pawl 30 abuts the wall delimiting the compartment 14 of the handle 12 when the drive member 20 is turned in the ratcheting direction. The cylindrical body 47 further includes two engaging faces or portions 43 and 44, one of which presses against an associated one of the inclined faces 341 and 342 of the pawl ~~[[34]]~~ **30**, as shown in Figs. 3A and 3B. This provides a more reliable support for the pawl 30.

Page 8, line 15 through page 9, line 4 have been amended as follows:

Referring to Fig. 4, in use, a portion of a fastener-driving tool, e.g., a shank 51 of a screwdriver 50 is inserted into the drive member 20 until an end face of the shank 51 is stopped by the inner flange 26 (i.e., the stop). The shank 51 of the screwdriver 50 is retained in the **inner periphery 22 functioning as an** engaging portion **[[22]]** of the drive member 20 by the retainer 24. When tightening a fastener 60, referring to Fig. 5, the user may grasp and turn the flange 25 rapidly, which causes rapid rotation of the drive member 20 and the shank 51 of the screwdriver 50. Thus, the fastener 60 is quickly turned in the tightening direction until a relatively large force is required for securely tightening the fastener 60. This is because the force required for turning the drive member 20 is smaller at the first stage of tightening the fastener 60. Another reason allowing rapid turning of the drive member 20 is that the flange 25 has an outer diameter that is much smaller when compared to the arm of force for turning the handle 12. Thus, the time for turning the fastener 60 to an almost tightened position is much shorter when compared to the use of the handle 12, as the angular travel of the drive member 20 is much shorter than that of the handle 12.

Page 9, line 19 through page 10, line 3 have been amended as follows:

Figs. 7 through 9 illustrate a second embodiment of the ratcheting wrench in accordance with the present invention, wherein like reference numerals designate like elements. In this embodiment, an annular groove 16 is defined in the inner periphery delimiting the hole 13 of the head 11, and a retainer 28', e.g., a C-clip is partially received in the annular groove 16 of the hole 13 and partially received in the annular groove (now designated by 27') of the drive member **[[20]] 20'**. Further, the flange (now designated by 25') of the drive member **[[20]] 20'** is formed on the other end of the drive member **[[20]] 20'**. It is noted that the drive member **[[20]] 20'** has an upper end that is flush with the upper end face of the head 11. Operation of the wrench of Figs. 7 through 9 is substantially the same as that of the wrench of Figs. 1 through 6.

Page 10, lines 13-15 have been amended as follows:

Preferably, the flange 25, 25' protrudes in a radial direction of the head 11 to a position beyond an end face of the head 11, allowing easy grasp and turning of the drive member **[[20]] 20'**.

Page 10, line 16 through page 11, line 4 have been amended as follows:

Fig. 11 illustrates a fourth embodiment that is modified from the second embodiment of the ratcheting wrench in accordance with the present invention, wherein like reference numerals designate like elements. In this embodiment, the lower end and the upper end of the drive member [[20]] 20' are located outside the head 11. In addition to the flange 25' formed on the lower end of the outer periphery of the drive member [[20]] 20', the upper end of the outer periphery of the drive member [[20]] 20' includes a shoulder 70. Further, an annular groove 72 is defined in the upper end of the outer periphery of the drive member [[20]] 20' in a position above the shoulder 70. A ring 74 is mounted around the upper end of the outer periphery of the drive member [[20]] 20' and has a side abutting against the shoulder 70. A retainer 76 is partially received in the annular groove 72, with the exposed portion of the retainer 72 abutting against the other side of the ring 74. Thus, the ring 74 is retained in place and acts as a member allowing the user to grasp for performing quick tightening/loosening functions and minor adjusting functions mentioned above.

Page 11, lines 5-12 have been amended as follows:

Fig. 12 illustrates a fifth embodiment of the invention that is modified from the fourth embodiment, wherein the flange 26 of the fourth embodiment is replaced with an annular groove 80 in the inner periphery 22 of the drive member [[20]] 20', and a retainer 78, such as a C-clip is partially received in the annular groove 80. Namely, the retainer 78 extends inward from the inner periphery 22 of the drive member [[20]] 20' to act as a stop for preventing the shank 51 of the screwdriver 50 from falling out of the drive member [[20]] 20' via the upper end of the drive member [[20]] 20' (c.f. Fig. 10).

Page 11, lines 13-17 have been amended as follows:

Fig. 13 illustrates a sixth embodiment that is modified from the fifth embodiment of the ratcheting wrench in accordance with the present invention, wherein like reference numerals designate like elements. In this embodiment, the annular groove 72 and the retainer 76 are omitted. Further, the ring 74 is fixed in place by ~~means of~~ riveting.

Page 11, lines 17-22 have been amended as follows:

Fig. 14 illustrates a seventh embodiment that is modified from the fourth embodiment of the ratcheting wrench in accordance with the present invention, wherein

like reference numerals designate like elements. In this embodiment, the annular groove 72 and the retainer 76 are omitted. Further, the ring 74 is fixed in place by ~~means of~~ riveting.

Page 11, lines 23-26 have been amended as follows:

The ratcheting wrenches of Figs. 11 through 14 allow easy manufacture and assembly, as neither the inner periphery of the hole 13 of the head 11 nor the outer periphery of the drive member 20, 20' is required to form an annular groove. The manufacturing cost is thus reduced.

Page 12, lines 1-8 have been amended as follows:

It is noted that the ratcheting mechanism and the switch member 40 are not limited to those disclosed herein and shown in the accompanying drawings. They can be replaced with any other structures allowing reversible or irreversible ratcheting operation. The "fastener-driving member" as used herein is not limited to the whole tool. Namely, the "fastener-driving member" may be a whole screwdriver or the like, a screwdriver shank 51 with a bit 52, or a screwdriver bit. Of course, ~~other~~ another member that serves the function of driving fasteners can be used as the fastener-driving member without departing from the scope of the invention.

Page 12, lines 9-19 have been amended as follows:

According to the above description, it is appreciated that the drive member 20, 20' of the ratcheting wrench in accordance with the present invention can be turned quickly such that the time for tightening/loosening a fastener can be significantly reduced. Further, the tightening force for the fastener can be finely adjusted. These advantages are provided by the flange 25, 25' on an end of the drive member 20, 20'. Further, the fastener-driving member 50 is securely retained in place by the retainer ~~[[23]]~~ 24, and the fastener-driving member ~~[[50]]~~ is prevented from disengaging from the drive member 20, 20' by a stop (i.e., the retainer 29' in Fig. 10 or the inner flange 26 in Fig. 3). Further, manufacture and assembly of the ratcheting wrench in accordance with the present invention can be simplified when the designs of Figs. 11 through 14 are adopted.